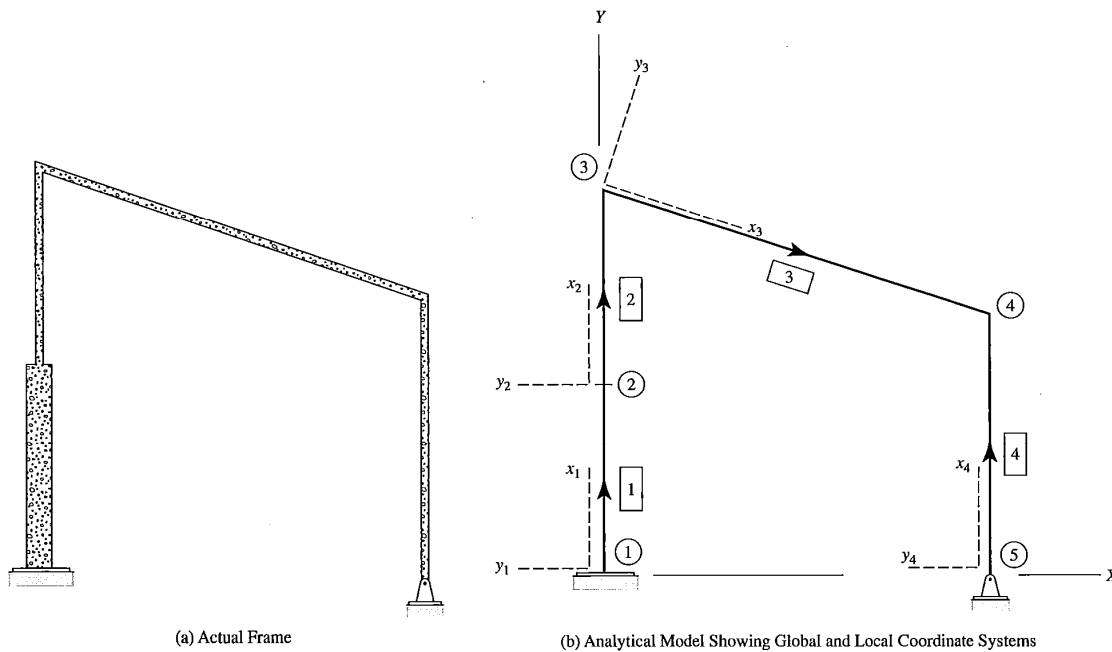


Chapter 5. Matrix Analysis of Plane Frames

5.1. Analytical Model (Kassimali §6.1)

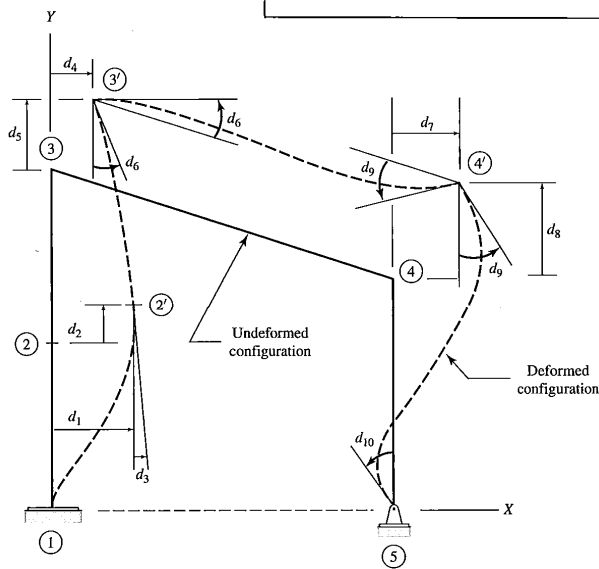
- A plane frame is divided into members and joints so that: (a) all members are straight and prismatic; (b) all external reactions act only at the joints.



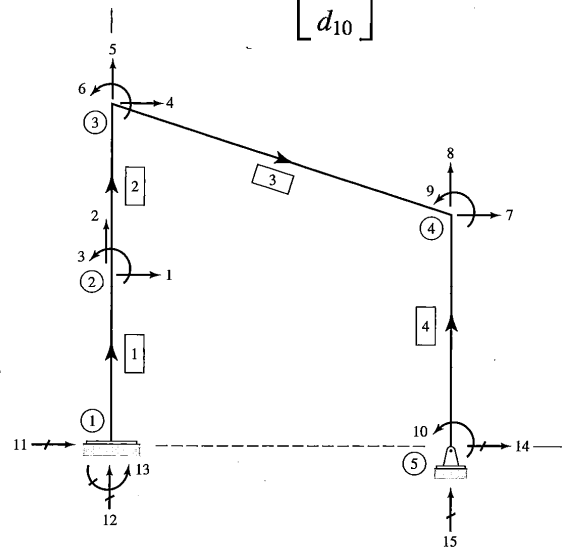
- A free joint has three DOFs, two translations and one rotation.
- Three coordinates (free and/or restrained) need to be defined at each joint: $NCJT = 3$
- Joint displacement vector \mathbf{d} is $NDOF \times 1$.
- Restrain coordinates: $NDOF + 1$ through $3(NJ)$.

$$\left. \begin{array}{l} NCJT = 3 \\ NDOF = 3(NJ) - NR \end{array} \right\} \text{ for plane frames}$$

$$\mathbf{d} = \begin{bmatrix} d_1 \\ d_2 \\ \vdots \\ d_9 \\ d_{10} \end{bmatrix}$$

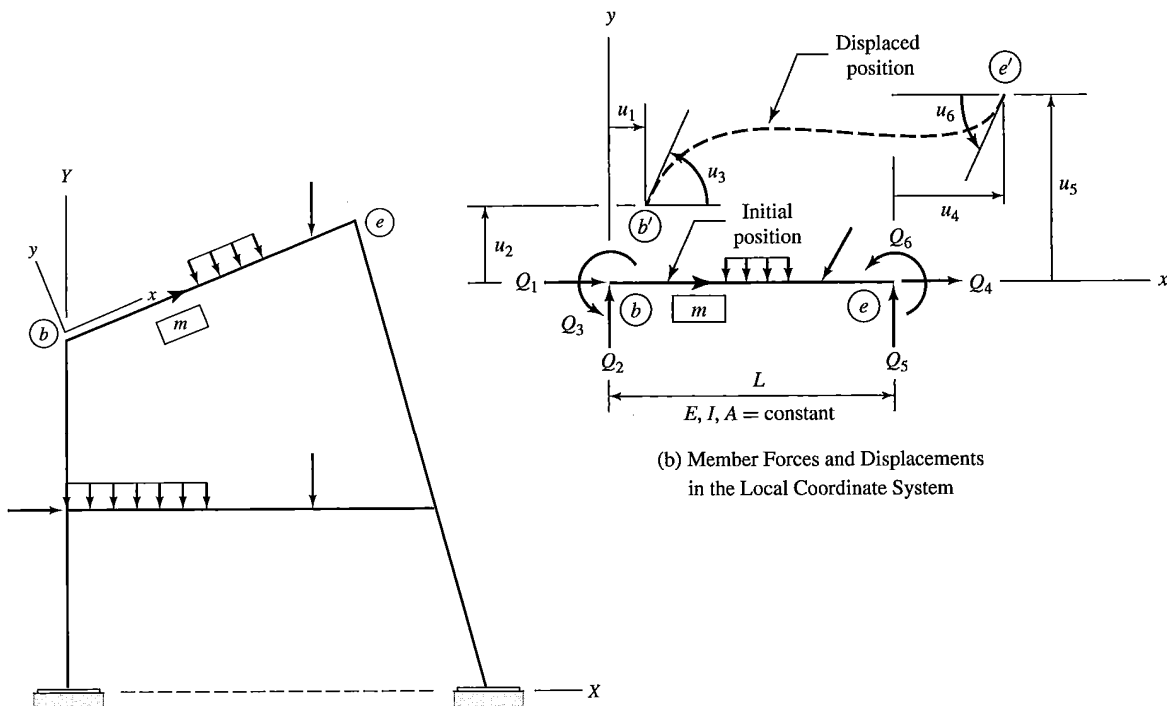


(c) Degrees of Freedom



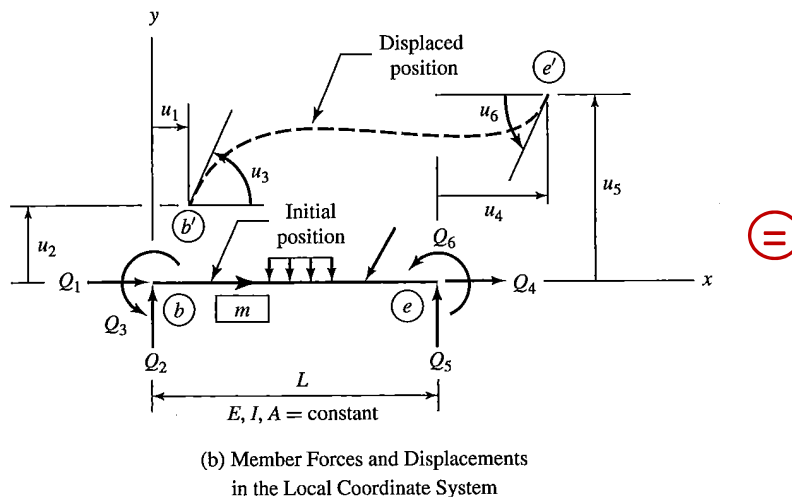
(d) Structure Coordinate Numbers
($NDOF = 10, NR = 5$)

5.2. Member Stiffness Relations in Local Coordinates (Kassimali §6.2)

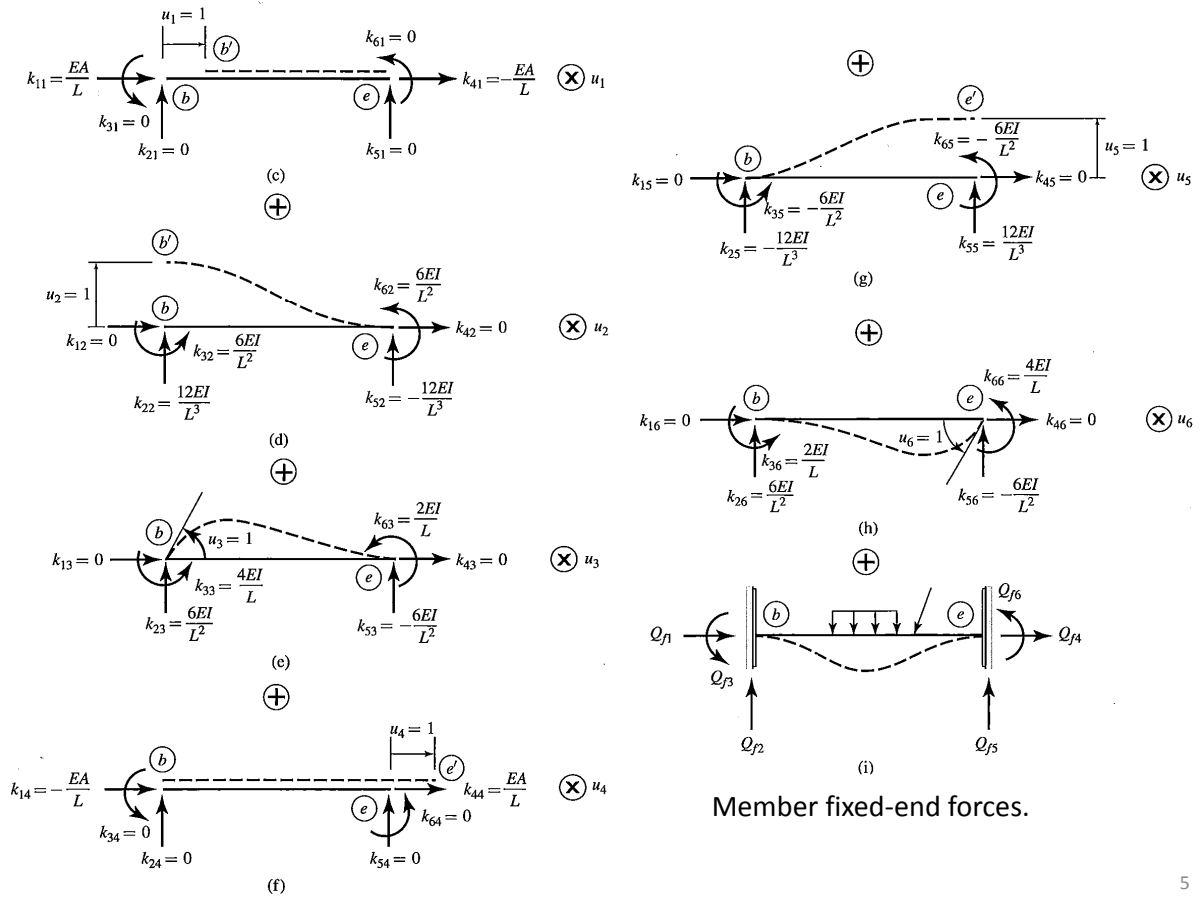


3

- Six member end displacements (rotations): $u_1 \sim u_6$
- Six member end forces (moments): $Q_1 \sim Q_6$
- Translations/forces in the positive directions of the local x and y axes are positive
- **Counterclockwise rotations/moments are positive**
- A member's local end displacements and end forces are numbered by beginning at its end b , with the translation/force in the x direction numbered first, followed by the translation/force in the y direction, and then rotation/moment.
- The displacements and end forces at the opposite end e are then numbered in the same sequential order.



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$$Q_i = \sum_{j=1}^6 (k_{ij}u_j) + Q_{fi} \quad i = 1, 2, \dots, 6$$

the stiffness coefficient k_{ij} represents the force corresponding to Q_i due to a unit value of the displacement u_j , and Q_{fi} denotes the fixed-end force corresponding to Q_i due to the external loads acting on the member.

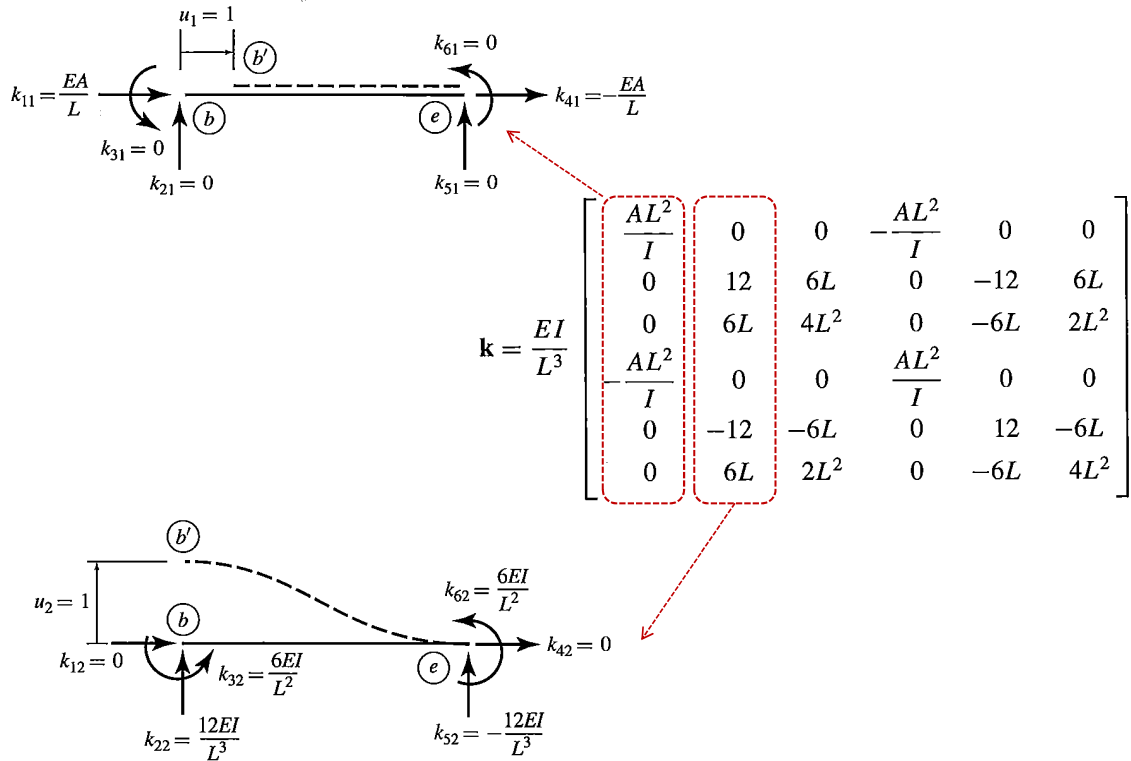
$$\begin{bmatrix} Q_1 \\ Q_2 \\ Q_3 \\ Q_4 \\ Q_5 \\ Q_6 \end{bmatrix} = \begin{bmatrix} k_{11} & k_{12} & k_{13} & k_{14} & k_{15} & k_{16} \\ k_{21} & k_{22} & k_{23} & k_{24} & k_{25} & k_{26} \\ k_{31} & k_{32} & k_{33} & k_{34} & k_{35} & k_{36} \\ k_{41} & k_{42} & k_{43} & k_{44} & k_{45} & k_{46} \\ k_{51} & k_{52} & k_{53} & k_{54} & k_{55} & k_{56} \\ k_{61} & k_{62} & k_{63} & k_{64} & k_{65} & k_{66} \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \\ u_3 \\ u_4 \\ u_5 \\ u_6 \end{bmatrix} + \begin{bmatrix} Q_{f1} \\ Q_{f2} \\ Q_{f3} \\ Q_{f4} \\ Q_{f5} \\ Q_{f6} \end{bmatrix}$$

$$Q = ku + Q_f$$

6x1 member fixed-end
force vector

6

- Member local stiffness matrix **k**



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$$\mathbf{Q} = \mathbf{k}\mathbf{u} + \mathbf{Q}_f$$

- Similar to member end forces in local coordinate system, **Q**, a member's local fixed-end forces, **Q_f**, are positive when in the positive directions of the local x and y axes, and the local fixed-end moments are considered **positive when counterclockwise**.

FIXED-END FORCE EXPRESSIONS		
Fixed-End Moments, Shears and Axial Forces for Various Loading Conditions		
No.	Loading	Equations for Fixed-End Moments, Shears, and Axial Forces
1.		$FS_b = \frac{Wl_2^2}{L^3}(3l_1 + l_2)$ $FM_b = \frac{Wl_1l_2^2}{L^3}$ $FS_e = \frac{Wl_1^2}{L^3}(l_1 + 3l_2)$ $FM_e = -\frac{Wl_1^2l_2}{L^3}$
2.		$FS_b = -\frac{6Ml_1l_2}{L^3}$ $FM_b = \frac{Ml_2}{L^3}(l_2 - 2l_1)$ $FS_e = \frac{6Ml_1l_2}{L^3}$ $FM_e = \frac{Ml_1}{L^3}(l_1 - 2l_2)$
3.		$FS_b = \frac{wL}{2} \left[1 - \frac{l_1}{L^4}(2L^3 - 2l_1^2L + l_1^3) - \frac{l_2^2}{L^4}(2L - l_2) \right]$ $FM_b = \frac{wL^2}{12} \left[1 - \frac{l_1}{L^4}(6L^2 - 8l_1L + 3l_1^2) - \frac{l_2^3}{L^4}(4L - 3l_2) \right]$ $FS_e = \frac{wL}{2} \left[1 - \frac{l_1}{L^4}(2L^3 - 2l_1^2L + l_1^3) - \frac{l_2^2}{L^4}(2L - l_2) \right]$ $FM_e = -\frac{wL^2}{12} \left[1 - \frac{l_1}{L^4}(4L - 3l_1) - \frac{l_2^3}{L^4}(6L^2 - 8l_2L + 3l_2^2) \right]$

No.	Loading	Equations for Fixed-End Moments, Shears, and Axial Forces
4.		$FS_b = \frac{w_1(L-l_1)^3}{20L^3} \left\{ (7L+8l_1) - \frac{l_2(3L+2l_1)}{(L-l_1)} \right.$ $\times \left[1 + \frac{l_2}{L-l_1} + \frac{l_2^2}{(L-l_1)^2} \right] + \frac{2l_2^3}{(L-l_1)^3} \left. + \frac{w_2(L-l_1)^3}{20L^3} \left\{ (3L+2l_1) \left[1 + \frac{l_2}{L-l_1} + \frac{l_2^2}{(L-l_1)^2} \right] - \frac{l_2^3}{(L-l_1)^3} \left[2 + \frac{15L-8l_2}{L-l_1} \right] \right. FM_b = -\frac{w_1(L-l_1)^3}{60L^3} \left\{ 3(L+4l_1) - \frac{l_2(2L+3l_1)}{L-l_1} \right. \times \left[1 + \frac{l_2}{L-l_1} + \frac{l_2^2}{(L-l_1)^2} \right] + \frac{3l_2^3}{(L-l_1)^3} \left. + \frac{w_2(L-l_1)^3}{60L^3} \left\{ (2L+3l_1) \left[1 + \frac{l_2}{L-l_1} + \frac{l_2^2}{(L-l_1)^2} \right] - \frac{3l_2^3}{(L-l_1)^3} \left[1 + \frac{5L-4l_2}{L-l_1} \right] \right. FS_e = \left(\frac{w_1+w_2}{2} \right) (L-l_1-l_2) - FS_b FM_e = \frac{L-l_1-l_2}{6} [w_1(-2L+2l_1-l_2) - w_2(L-l_1+2l_2)] + FS_b(L) - FM_b $
5.		$FA_b = \frac{WL}{2}$ $FA_e = \frac{WL}{2}$
6.		$FA_b = \frac{w}{2L}(L-l_1)(L-l_1-l_2)$ $FA_e = \frac{w}{2L}(L-l_1-l_2)(L+l_1-l_2)$
7.		$FT_b = \frac{Ml_2}{L}$ $FT_e = \frac{Ml_1}{L}$

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